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You are here: [Home](#) » [News](#) » **Aisin-Warner 09G Electronics – Anatomy of Shift Controls**

Aisin-Warner 09G Electronics – Anatomy of Shift Controls

The Aisin-Warner 09G six speed FWD transmission, or TF-60, has increased in popularity and technicians need to become more familiar with all aspects. The 09G can be found in a variety of Volkswagen FWD vehicle platforms produced after 2003. Its close cousins, the TF80 (Volvo and GM) and TF81 (Ford and Mazda) further enhance a huge market potential for the transmission repair industry.

Much focus has been spent over the years on valvebody wear issues and other deficiencies of the 09G as well as the TF80 / TF81 models. Once a repair had been made, shift or apply issues still existed at times requiring more attention to the controls.

The experts at Rostra have devoted a substantial amount of effort to determine how all of the aspects of Aisin 6-speed transmissions (valvebody, solenoids, and TCM) work together to provide a good functioning and durable transmission repair.

The following is a summary of that research:

Aisin-Warner has made several improvements related to the operation and design of this 6-speed family relative to earlier 5-speed (AW55-50) family, including the utilization of dedicated linear solenoids to directly control clutches. With this improved system, the TCM has more ability to adapt and precisely control the quality of shifts but ultimately the control is limited. Whenever any conditions change within the hydraulic circuit, such as the valvebody or solenoids, adjustments to the system may be necessary. This article will dissect exactly what is happening during select shifts and how to make adjustments to the valvebody in order to obtain the desired level of drivability. The 09G will be the specific focus of the article although most of the information holds true for the 09D (TR60), TF80, and TF81.

First, let's examine the solenoids and their role in the transmission. The 09G has six linear solenoids and two on/off solenoids. Of the six linear solenoids, four of the units directly control clutches and brakes via spring loaded spool valves in the valvebody. The remaining two control line pressure and torque converter lock-up. Five of the linear solenoids are normally open and function to reduce pressure output with increased current draw. The TCC solenoid is normally closed and increases pressure output with increased current draw. The two on/off solenoids are normally closed and are energized for various purposes including engine braking and torque converter lock-up. The detailed function of each solenoid and clutch is illustrated in Figure 1.

	On/Off Solenoids		Linear Solenoids						Clutches and Brakes				
	N88 (SS1)	N89 (SS2)	N92 (K1 Control)	N282 (K2 Control)	N90 (K3 Control)	N283 (B1 Control)	N93 (EPC)	N91 (TCC)	K1 Clutch	K2 Clutch	K3 Clutch	B1 Brake	B2 Brake
Park													
Reverse													
Neutral													
1st													
2nd													
3rd													
4th													
5th													
6th													
	Solenoid Energized or Clutch/Brake Engaged								Solenoid on During Lock-up				
	Solenoid Energized or Clutch/Brake Engaged in Tiptronic Mode Only								Solenoid Modulated to Control Lock-up Condition				
	Solenoid Modulated Depending on State of Throttle and Transmission								Solenoid Energizes During Shifts				

Figure 1 09G Solenoid and Component Chart

Operation of Solenoids During Shifts

To understand how the linear solenoids control the engagement of the clutches and brakes, we must first understand the signal driving them. The linear solenoids are supplied 12V power in the form of a 300 Hz PWM signal. An accepted but ineffective aftermarket industry standard is to simply measure current being supplied to the solenoids to determine load. Due to the variations in current draw depending on heat and specific resistance of the solenoid, measuring duty cycle percentage is a more consistent, accurate, and preferred representation of solenoid load. The solenoids are driven by signals that vary from 12-60% duty cycle with 60% being “fully energized,” limiting the normally open solenoid’s output to near 0 psi. At 12% duty cycle the solenoid is essentially “off” which allows all input pressure to pass through a normally open solenoid. The majority of pressure modulation occurs between 20 and 55% duty cycle. Figure 2 below shows the signals applied to K1, K2, and K3 control solenoids on the tail-end of a 3-4 shift where the duty cycles are 12%, 40% and 60% respectively. As a reference, the time elapsed between each peak is 3.33 milliseconds.

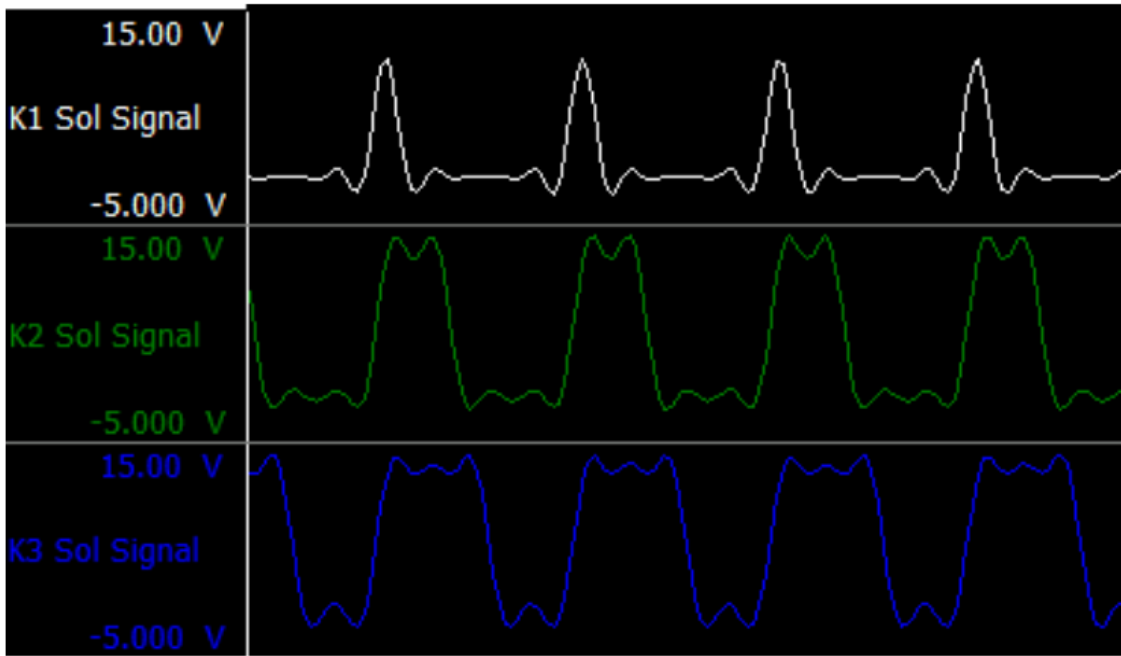


Figure 2 Raw Signal Data from 09G

Electronic control over the entire pressure range of the transmission means that the TCM can ramp the rise and drop of pressure to specific clutches and brakes in order to achieve a smooth shift. Most shifts are comprised of one clutch (or brake) being applied and another clutch (or brake) is being released (synchronous shift). In an ideal shift situation, the two components will have minimal time between application of the two components. Figure 3 illustrates how the signals and pressures are being modulated in the same 3-4 shift as featured in Figure 2. Figure 3 shows that during the 3-4 upshift the K2 clutch is applying and the K3 clutch is releasing. Notice how K3 control solenoid ramps up the signal to inversely drop the pressure in the K3 circuit while at the same time, the opposite occurs for the K2 solenoid signal and hydraulic circuit. Application pressures for the K2 and K3 clutches are near 30 psi indicating that for approximately 100 ms, neither the K2 nor K3 clutch was engaged. If the time between clutch applications was longer (over ¼ second) the RPMs would begin to rise as if in neutral then drop quickly resulting in a shift flare then bump. On the other hand, if the K2 clutch is applied before K3 releases; the transmission is effectively trying to temporarily operate at two different gears simultaneously resulting in a bind-up.

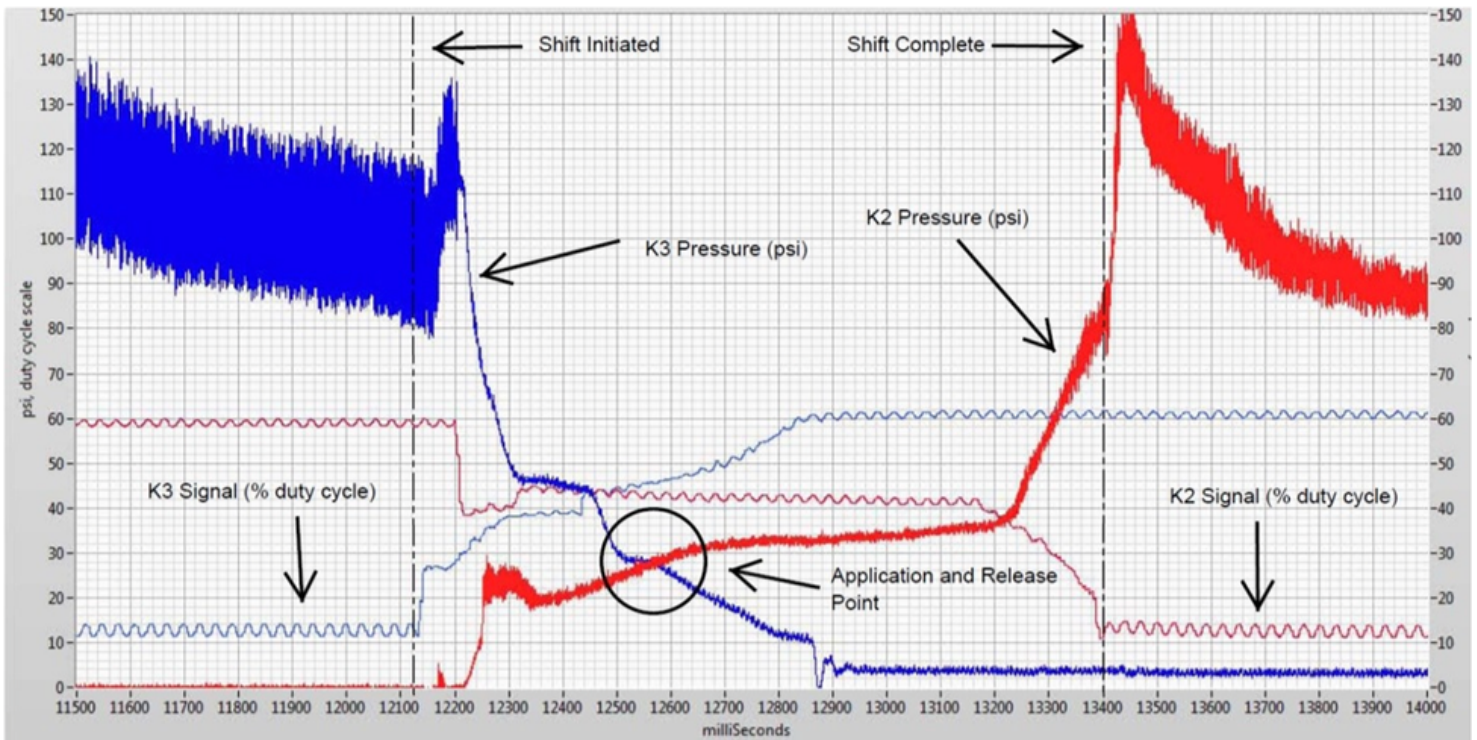


Figure 3 Signals and Pressure 3-4 Shift

Relearn and Calibration

Unlike the solenoids in the AW55 transmission, the AW 6 solenoids are not adjustable. Aisin-Warner has elected to compensate for the variations in solenoid output by implementing adjustment screws directly on the valvebody. Calibration of the valvebody with solenoids installed as a whole system is necessary to ensure a consistent quality of shifts. In the event that any clutch's control system output relative to drive signal changes, shift quality and drivability may suffer resulting in flares or bind-ups. Poor shift performance could be caused by simply replacing solenoids with different preset calibrations or repairing the valvebody to fix leaks which the TCM has already taken into account. In most situations the TCM is capable of adjusting the drive signals to smooth out any harsh shifts. Through normal driving, this may be a slow and tedious process requiring up to 50 miles of drive time. To help accelerate this process, it is recommended that the shop perform a relearn procedure:

1. With the transmission at normal operating temperature and no codes present, shift from Park to Reverse to Neutral to Drive and back, staying in each range for 10 seconds. Repeat this at least 5 times.
2. Accelerate through the first 5 gears under light throttle (10%) then come to a stop. Repeat this 5 times.
3. Accelerate through all gears at medium throttle (50%) then come to a stop. Repeat this 5 times.
4. Manually downshift through all gears. Repeat this 5 times.

It's important to remember that the intent of the relearn procedure is to simulate many miles of average driving in a short time. If there is one particular shift that is harsh, it would be beneficial to repeat that shift with greater frequency.

After performing this relearn procedure, it is still possible for the hydraulics to be so far from the system's norm that manual calibration of the valvebody is necessary to achieve the desired drivability. The first step in calibrating a valvebody is to identify which clutch or brake control needs to be adjusted. Figure 4 below illustrates common drivability issues and the appropriate clutch and brake control to adjust in order to resolve the issue. For example, if a vehicle was experiencing a flare on the 3-4 shift, the resolution would be to turn the adjustment screw on the spring tensioner for the spool valve that shares the bore with the K2 control solenoid in the counter-clockwise direction. One complete turn of the screw can resolve a slight application issue while an obvious flare or bind-up may be resolved by two complete rotations of the screw.

Component	Turn Screw Clockwise (Decrease Pressure)	Turn Screw Counter-Clockwise (Increase Pressure)
B1	<ul style="list-style-type: none"> • 1-2 Bind-up • 5-6 Bind-up • 3-2 Bump 	<ul style="list-style-type: none"> • 1-2 Flare • 5-6 Flare • 3-2 Flare
K1	<ul style="list-style-type: none"> • Quick Harsh Drive Engagement • 5-4 Bump 	<ul style="list-style-type: none"> • Delayed Harsh Drive Engagement • 5-4 Flare
K2	<ul style="list-style-type: none"> • 3-4 Bind-up 	<ul style="list-style-type: none"> • 3-4 Flare
K3	<ul style="list-style-type: none"> • Quick Harsh Reverse Engagement • 2-3 Bind-up • 4-5 Bind-up • 4-3 Bump • 6-5 Bump 	<ul style="list-style-type: none"> • Delayed Harsh Reverse Engagement • 2-3 Flare • 4-5 Flare • 4-3 Flare • 6-5 Flare

Figure 4 Clutch and Brake Controller Adjustment Chart

The engagement into Drive and Reverse can be particularly tricky to diagnose whether the pressure is too high or low. In Figure 5, the effects of both high and low K3 pressures can be seen on the engagement of reverse. In an ideal clutch application, the pressure gradually rises to 30 psi as seen in the Smooth Engagement. If the pressure rise is too sudden, the engagement will be harsh. Notice how the early engagement reaches the engagement pressure before the ramp sequence while the delayed engagement finishes the ramp sequence before it reaches the engagement pressure. The harshness of both the early and delayed engagement featured in Figure 5 could be perceived to be of the same magnitude. When diagnosing the difference between high and low pressure, it is important to focus on the time between moving the gear selector and the bump that follows. The early engagement will happen almost instantaneously whereas the delayed engagement will take up to 2 seconds to feel the bump.

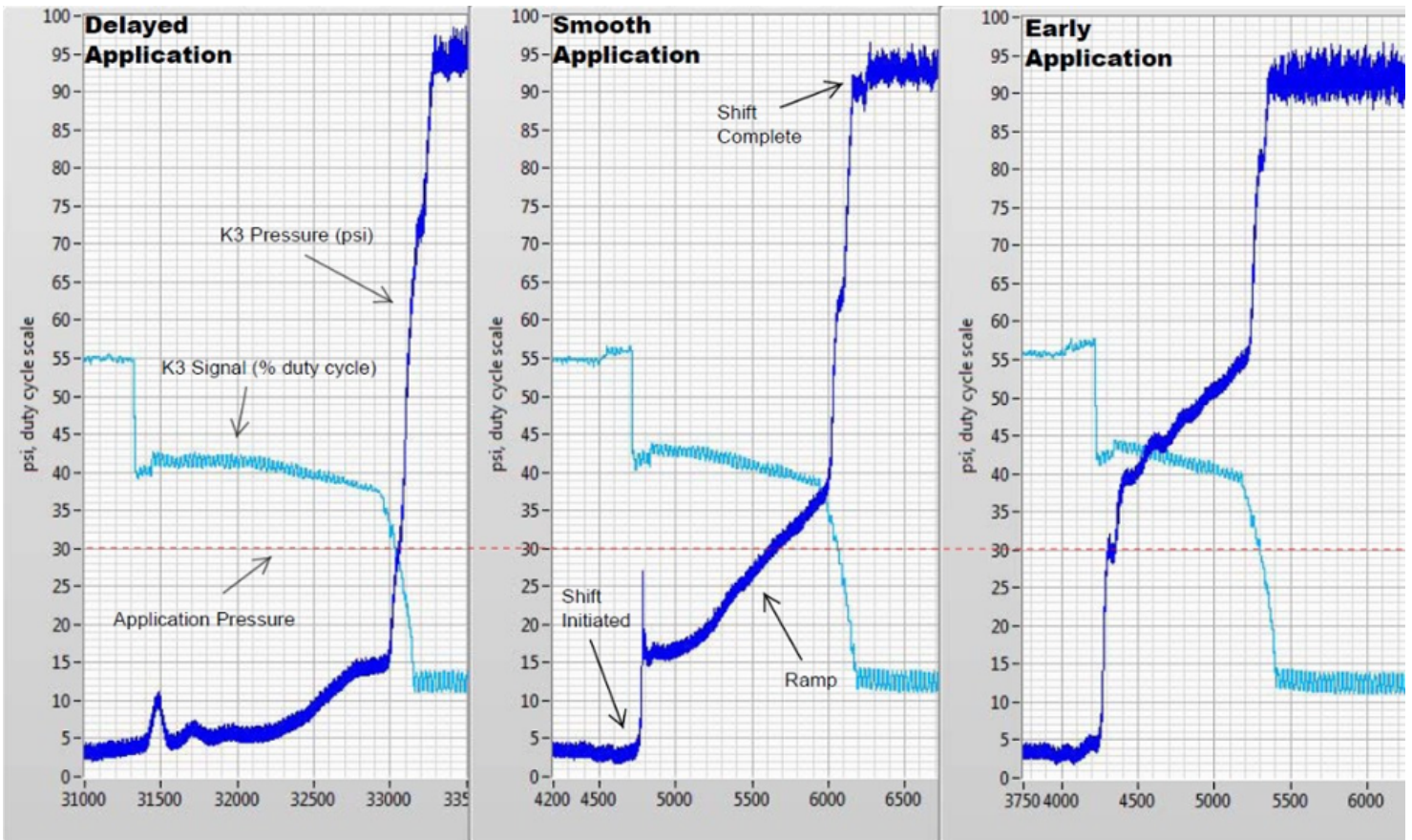


Figure 5 Delayed, Smooth and Early Reverse Engagements

The line pressure circuit and EPC solenoid do not have adjustment screws and all of their corrections are relegated to the TCM. The TCC solenoid does have an adjustment screw that is typically staked and nonadjustable. Similarly, TCC corrections can only be made by the TCM. It should be noted that if the output pressure is lower than normal in the TCC solenoid, the initial torque converter lock-up engagement while in third gear could be firm and feel similar to a 3-4 shift. When diagnosing it is important to confirm which gear is engaged by using a scan tool or the Tiptronic system of the vehicle. Shift firmness should work itself out through the relearn procedure.

The improved design of the hydraulic control system in the 09G allows you to take the guess work out of resolving drivability issues. Direct control of the clutch and brake components through dedicated solenoids, along with the ability to adjust pressures of the hydraulic circuits, make diagnosis and resolution of drivability issues more of a science and less of an art.

Currently, OEM solenoids are only available by purchasing an entire valvebody. Rostra however, has linear solenoids available and with an improved design. Problems with OE solenoids involve wear or debris build up affecting the internal bushings. Rostra has redesigned the bushings to prevent those issues such as wear or sticking of the spool valve. The solenoid body has also been redesigned for better oil flow. The on-off solenoids are under development as well. Contact Rostra Powertrain or [Rostra distributors](#) for a complete listing of Aisin solenoids as well as other Rostra products.

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